

Localization of Damage in Beams Using Interferometric Techniques

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ABSTRACT

Two interferometric techniques and their applications in structural damage identification are presented in this paper. Out-of-plane displacement fields of modal response are measured with a pulsed electronic speckle pattern interferometric system (ESPI). The modal rotation fields, defined as the spatial derivative of the displacement field, are measured with a pulsed speckle shearography system. The measurements using these two interferometric systems are compared with measurements from experimental modal analysis and results from finite element analysis. This comparison shows that these two interferometric techniques, which allow non-contact, full-field measurements, are well suited to the measurement of modal response. A damaged beam with free-free boundary conditions is analyzed. The damages studied are small cuts perpendicular to the beams longitudinal axis. The bending moments and shear forces, which are related to the second and third order spatial derivatives of the modal displacement field, in the undamaged and damaged states are computed using numerical differentiation techniques. The damage is localized by looking for maximum values and/or perturbations of damage indicators based on bending moments and shear forces along the beam. The pulsed speckle shearography system leads to a significant improvement in the computation of bending moments and shear forces and, therefore, better damage localizations than the ones obtained with the pulsed ESPI system.

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INTRODUCTION

Structural damage identification based on vibration characteristics, which encompasses detection, localization and quantification, is a very active research topic. Thus, many methods can be found in the literature [1]. One of the most successful and well established is the mode shape curvature method, presented by Pandey *et al.* [2]. It has been improved [3-4], generalized to frequency response function data [5] and applied to composite structures [6-9]. Usually the experimental data used are sparse displacement fields that need to be numerically differentiated. The differentiation of these sparse measurements gives raise to propagation and amplification of experimental noise and, therefore of poor damage localizations. One way to improve the damage localization is to use full-field experimental techniques.

In this paper, two full-field, non-contact and high resolution optical techniques are presented, validated and applied to localize damage in a beam. The first one is the electronic speckle pattern interferometry (ESPI) and allows the measurement of displacement fields. Speckle shearography is used to obtain the spatial derivatives of displacement, i.e the rotation field. These techniques have their origins in holographic interferometry [10]. They are based on the comparison of two patterns or phase maps generated by the coherent superposition of two wavefronts. Each of these phase maps refers to a state of an object with a rough surface. Considering that the first phase map corresponds to the undeformed state and the second one to the deformed state, it is possible to measure the motion of the objects surface. The modal response of an object can be assessed by using a pulsed laser in the experimental set-up of these techniques.

Two experimental set-ups were implemented to measure the stationary harmonic motion at the first two natural frequencies of free-free beams, acoustically excited, with the ESPI and shearography techniques. These measurements are compared with those obtained using standard experimental modal analysis and the finite element method. Two damage indicators, based on the distribution of bending moments and shear forces, are presented and applied to the damage localization of two cuts in an aluminium beam. Since the bending moments are related to the second and third derivatives of the displacement, the data from the ESPI measurements must be differentiated twice and thrice, respectively. However, when one uses the data obtained with the shearography system, the bending moment and shear force is computed respectively by differentiating once and twice the rotations. Because of this decrease in the order of the needed differentiations, which diminishes the propagation and amplification of noise, the pulsed speckle shearography system presents better damage localizations than the pulsed ESPI system.

VALIDATION OF INTERFEROMETRIC TECHNIQUES

Experimental modal analysis usually relies on the measurement of a sequence of frequency response functions (FRF), where the excitation force is applied using an impact hammer and the response is measured by an accelerometer at one edge of the beam. The use of the accelerometer adds mass to the beam and, depending on the ratio of the accelerometer mass to the beam mass and its location relative to the modal amplitude, it can significantly change the structural dynamic behavior. In fact,

preliminary measurements showed that the influence of the accelerometer mass in an aluminum beam is substantial. In order to minimize this problem, instead of an aluminum beam, a stainless steel beam was studied.

Experimental Modal Analysis

A stainless steel beam 450 mm long, 71 mm width and 10 mm thick was analyzed free in space. The free-free conditions were accomplished by suspending the beam with rubber bands. Transient type excitations were applied using an impact hammer and the negative exponential responses were measured by an accelerometer of 2.4 g. The excitations were applied at 31 points along the half width of beam, while the responses were measured at one of the beams end. This analysis was performed in a 0 Hz to 1 kHz frequency band, with a resolution of 156.25 mHz.

The natural frequencies, damping ratios and modal participation factors were obtained through the polyreference technique in the time domain. The corresponding mode shapes were obtained by least square fitting using the polyreference technique in the frequency domain [11-12]. This analysis was performed with the 31 measured FRFs. Two natural frequencies were identified in the frequency band and their respective mode shapes were constructed. The first two natural frequencies are 260.95 Hz and 718.72 Hz with damping ratios of 0.084% and 0.038%, respectively.

The modal model is created from the identified modal parameters above. This model should describe the vibrational behavior of the structure. The validation of this model is performed by comparing the synthesized and measured FRF and the modal assurance criteria (MAC) [12-13]. The MAC measures the level of orthogonality of different modes. Ideally, the diagonal elements of the MAC matrix should take a value of 100%, while the off-diagonal elements should be zero. In the present case, the diagonal elements have a value of 100.00% and the off-diagonal are 0.26%.

ESPI and Shearography

To perform dynamic measurements, the pulsed ESPI system, also known as pulsed TV holography, uses holographic interferometry principles to compare two states of an object in a very short time interval. The present system uses a double pulse Ruby laser to generate a pair of impulses with 690 nm wavelength, separated by 400 microseconds. The double exposure holograms are recorded by a 512 x 512 resolution CCD digital camera. These holograms contain information of the phase differences of two wave fronts.

The beam was suspended and excited acoustically at each natural frequency using a loudspeaker. At each frequency, the signal amplitude was adjusted in order to fit the amplitude of the vibration within the range of the measurement system. This signal is also used to synchronize the laser trigger and the harmonic vibration of the beam. As the beam is free in space, it presents both elastic and rigid body modes. The elastic mode shape is confined from the rigid body motion, which has small amplitude and lower frequency, i.e., a larger period, since the impulses are separated by only 400 μ s.

The spatial carrier, which is introduced in the holograms, is demodulated to obtain the primary phase map [14-16]. Since each of the holograms corresponds to an instant in time, the final phase map corresponding to the surface motion is computed by subtracting them. Figure 1 shows the phase maps of the displacement fields of the two

first mode shapes. These mode shapes, also presented in Figure 1, are obtained by post-processing the phase maps using filtering and unwrapping techniques [14, 17]. One can define the displacement field of a beam mode shape as a vector of norm one by taking the amplitudes of the longitudinal axis at half width of the beam. The displacement fields thus obtained and the ones obtained by experimental modal analysis and finite element analysis show an excellent agreement (Figure 2). Small discrepancies at the edge of the beam are observed between the experimental modal analysis and the finite element results, which may be related to the presence of the accelerometer.

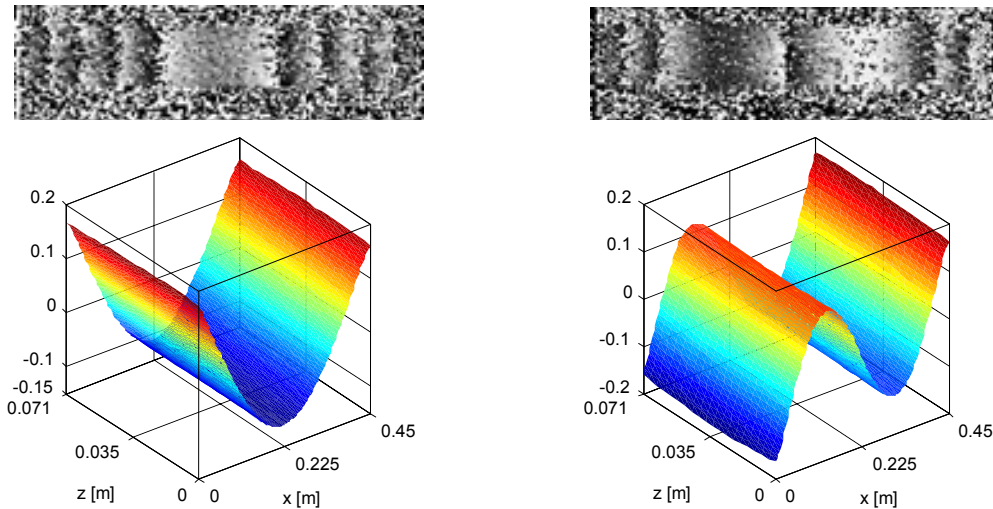


Figure 1. Phase maps and displacement fields of the first two mode shapes using the pulsed ESPI system.

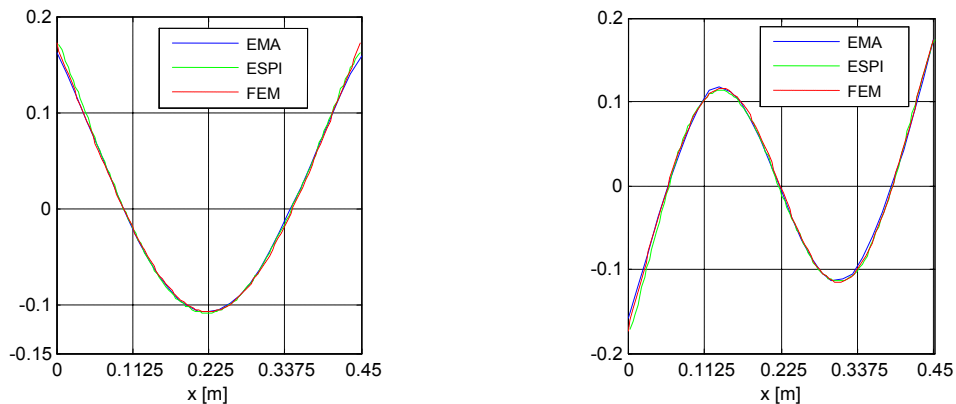


Figure 2. Displacement fields of the first two mode shapes using experimental modal analysis (EMA), the pulsed ESPI system, and the finite element method (FEM).

The pulsed speckle shearography system is based on a Mach-Zehnder interferometer and, like the ESPI system, uses a double pulsed laser and a CCD digital camera to register two holograms in two consecutive instants. The phase map relative to the rotation field, i.e. the displacement field gradient, is constructed based on these two holographic recordings.

Although the rotation field measurements were performed with a similar experimental set-up to the one used in the ESPI system, the synchronization signal of the excitation and the laser trigger was adjusted in order to frame the amplitude of

rotations into the measurement range. The shearography system is more insensitive to uniform displacements than the ESPI system, therefore allowing a better confinement of rigid body motions.

Figure 3 shows the phase maps of the rotation fields relative to the first two modes of the undamaged free-free beam. As in the ESPI technique, by post-processing these phase maps we obtain the vectors that define the corresponding rotation fields.



Figure 3. Phase maps of the first two mode shapes using the pulsed speckle shearography system.

These rotation fields are compared with the ones obtained by numerical differentiation of the experimental modal analysis displacement fields and the rotation fields computed by finite elements. There is a relatively fair correlation among the three kinds of rotation fields (Figure 4). Similarly to the ESPI results (see Figure 2), we observe some discrepancies between the modal and the finite element fields at the edge of the beam.

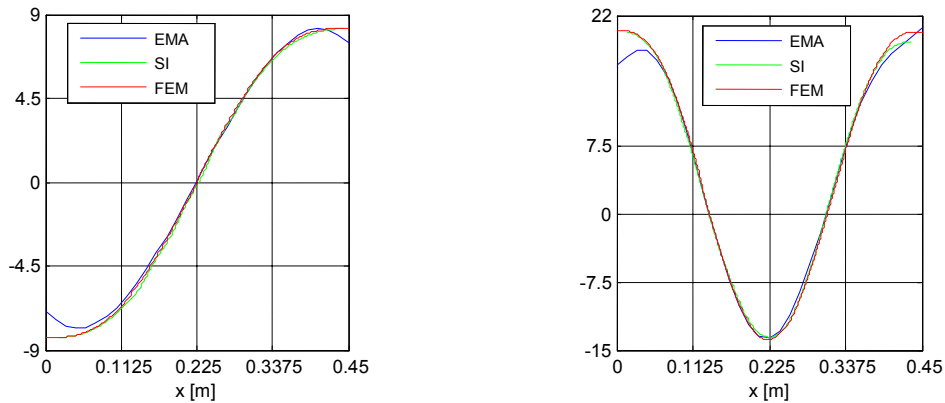


Figure 4. Rotation fields of the first two mode shapes using experimental modal analysis (EMA), the pulsed speckle shearography system (SI), and the finite element method (FEM).

DAMAGE LOCALIZATION

An aluminum beam was chosen as a test structure to localize damage. The first two natural frequencies of the beam are determined by classical experimental modal analysis procedures and the mode shapes are obtained using a pulsed ESPI and pulsed speckle shearography systems.

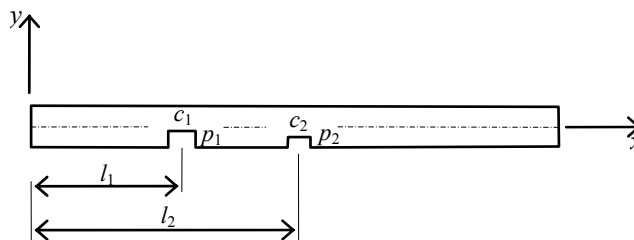


Figure 5. Free-free aluminum beam with two cuts.

The damage studied consists of two small cuts perpendicular to the beams longitudinal axis. The free-free aluminum beam analyzed is $L = 260$ mm long, $b = 45$ mm wide and $h = 6$ mm thick. The cuts are located at $l_1 = 75$ mm and $l_2 = 130$ mm with dimensions $p_1 = 0.65$ mm, $c_1 = 4$ mm, $p_2 = 0.75$ mm, $c_2 = 0.75$ mm (Figure 5).

The damage indicators used in this work are the bending moment indicator (BMI) and shear force indicator (SFI). These indicators are defined, respectively, by [18]:

$$BMI(x_l) = \left| \frac{\tilde{M}(x_l) - M(x_l)}{\max[M(x)] - \min[M(x)]} \right|, \quad SFI(x_l) = \left| \frac{\tilde{V}(x_l) - V(x_l)}{\max[V(x)] - \min[V(x)]} \right| \quad (1)$$

where the tilde denotes the damaged beam quantities and x_l is the coordinate where the indicator is computed.

The relations among the displacement and rotation fields, $y(x)$ and $\theta(x)$, the bending moment $M(x)$ and the transverse force $V(x)$ are defined considering the Euler-Bernoulli beam theory:

$$M(x) = EI \frac{d^2 y(x)}{dx^2}, \quad V(x) = EI \frac{d^3 y(x)}{dx^3} \quad \text{or} \quad M(x) = EI \frac{d\theta(x)}{dx}, \quad V(x) = EI \frac{d^2 \theta(x)}{dx^2} \quad (2)$$

The first two relations in (2) are used with the measurements made with the ESPI system, as this technique gives us the out-of-plane displacements, whereas the last two relations are applied to the shearography measurements.

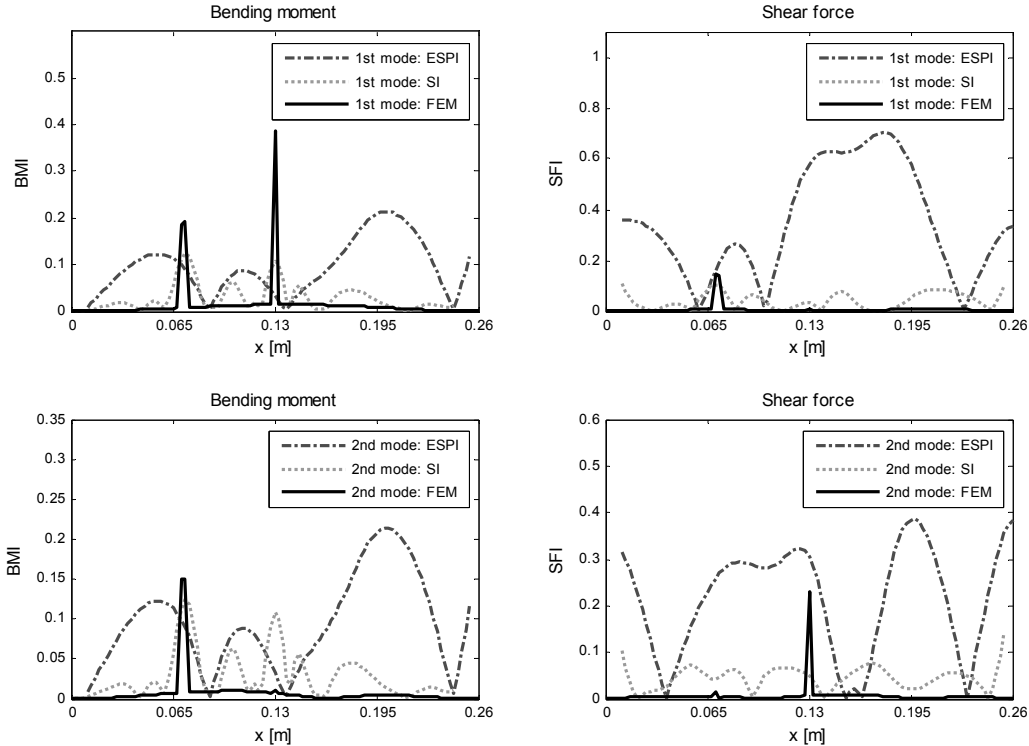


Figure 6. BMI and SFI values of the first and second mode shapes using pulsed ESPI and pulsed shear (SI) interferometric systems, and the finite element method (FEM).

The computed BMI and SFI indicators of the first and second mode shapes relative to the measurements of the pulsed ESPI system, the pulsed speckle shearography system and results from finite element analysis are presented in Figure 6. The best damage localization is the one obtained with the pulsed speckle shearography system and the BMI indicator. Besides showing a similar trend to the BMI distribution computed with finite element data, the BMI distribution computed with the pulsed speckle shearography measurements shows maximum values at the beam sections where the cuts are located. The cuts are not localized using the SFI indicators, because the shear force is obtained by differentiating twice the rotation field measured with the shearography system and thrice the displacement field measured with the ESPI system. Since these differentiations are performed by simultaneously applying signal differentiation and high frequency filtering techniques, the signal high frequencies due to damage, which are usually mixed up with high frequency measurement noise, are eliminated.

CONCLUSIONS

Two interferometric techniques and their applications to damage localization of cuts in beams were described in this paper. The measurements of the displacement and rotation fields using ESPI and speckle shearography systems, respectively, and subsequent application of a numerical differentiation method, allowed us to successfully determine the bending moments and shear forces of modal responses of a beam free in space. The reduction in one order of the spatial derivative needed to compute the bending moments and the shear forces by using the pulsed speckle shearography system leads to a significant improvement in the localization of damage.

ACKNOWLEDGEMENTS

The authors greatly appreciate the financial support of FCOMP-01-0124-FEDER-010236 through Project Ref. FCT PTDC/EME-PME/102095/2008.

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Proceedings of the 6th European Workshop - Structural Health Monitoring 2012, EWSHM 2012

Volume 1, 2012, Pages 198-205

6th European Workshop on Structural Health Monitoring 2012, EWSHM 2012; Dresden; Germany; 3 July 2012 through 6 July 2012; Code 102922

Localization of damage in beams using interferometric techniques

(Conference Paper)

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Abstract

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Reaxys Database Information

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ISBN: 978-394028341-2 **Source Type:** Conference Proceeding **Original language:** English

Document Type: Conference Paper

Sponsors: EMBRAER S. A., WOLFEL Beratende Ingenieure GmbH + Co. KG, Sachsisches Staatsministerium für Wissenschaft und Kunst

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