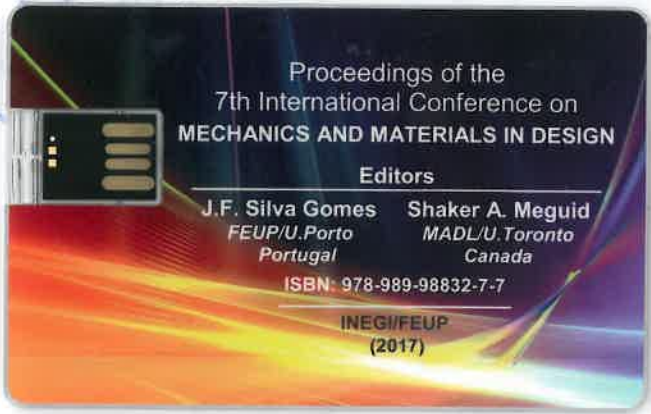


M2D2017

Albufeira, 11-15 June 2017





Proceedings of the
7th International Conference on
MECHANICS AND MATERIALS IN DESIGN

Editors

J.F. Silva Gomes <i>FEUP/U.Porto</i> <i>Portugal</i>	Shaker A. Meguid <i>MADL/U.Toronto</i> <i>Canada</i>
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Editors Preface

M2D2017 is the seventh international gathering of a prestigious series of conferences coordinated by the International Scientific Committee of Mechanics and Materials in Design. This series of conferences is wholly devoted to advances in mechanics, materials, structural integrity and design. M2D2017 is sponsored by the University of Porto, the University of Toronto and the University of Algarve. The conference attracted over 230 participants with 360 accepted submissions from 40 countries out of 416 submissions. These papers were presented in June 11-15, 2017 in the magnificent city of Albufeira/Algarve, Portugal. The conference themes which address novel and advanced topics in Mechanics and Materials in Design focused on computational mechanics, experimental mechanics, fatigue and fracture mechanics, composite and advanced materials, nanotechnologies and nanomaterials, tribology and surface engineering, mechanical design and prototyping, biomechanical applications, civil engineering applications, impact and crashworthiness, energy and thermo-fluid systems, and industrial engineering and management.

The conference also included an Open Forum on *The Challenges Facing Engineering Education*, where an expert panel with over 100 years of collective and active researchers and educators addressed the roles of professors that they meet, the obligations of their stakeholders and current challenges facing engineering education.

We believe that the meeting offered our delegates a forum for the dissemination of their recent work in mechanics and materials and their applications in engineering design, fostered research that integrates mechanics and materials in the design process, and promoted exchange of ideas and international co-operation among scientists and engineers in this important field of engineering.

We are particularly indebted to the authors and special guests for their presentations. Each of the more than 360 contributions offered opportunities for thorough discussions with the authors. Particularly, we acknowledge the excellent contributions of the participants, their innovative ideas and research directions, the novel modeling and simulation techniques, and the invaluable critical discussions. We are also indebted to the outstanding keynote speakers who highlighted the conference themes with their contributions. We also take this opportunity to thank the members of the International Scientific Committee, the members of the Advisory Committee and the reviewers for their time, effort and helpful suggestions.

We offer our sincere gratitude to the symposia organisers for their efforts and valuable contributions to the success of the event, and the local organising committee for attending to the conference demands and delegates needs.

All in all, M2D2017 was a great success and the credit must go to all the participants for their significant contributions and lively discussions, the keynote speakers for bridging the gap between the different disciplines and the organizing committee for an absolutely superb organization of the meeting in this magnificent city. To all of you, we offer our gratitude.

Given the rapidity with which science is advancing in all areas of mechanics and materials, the next conference in this series (Integrity, Reliability and Failure - IRF2018) will take place in Lisbon, the capital city of Portugal, in July 2018. Undoubtedly, we expect IRF2018 to be as stimulating and interesting as M2D2017, as evidenced by the excellent contributions offered in this current event. We look forward to seeing all of you in Lisbon in 2018.

Shaker A. Meguid and J.F. Silva Gomes

Albufeira / Portugal, June 2017

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PAPER REF: 6712

SHEET METAL FORMING BY NUMERICAL SIMULATIONS: 2D AND 3D AXISYMMETRIC DEFORMATION OF A CYLINDRICAL PIECE

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ABSTRACT

The aim of this work is conduct numerical simulations to verify the deep drawing process and the shape of the final stamping component of a simple profile of a sheet metal geometry. To evaluate the design of sheet metal forming, ANSYS software was used. A nonlinear dynamic explicit numerical model was developed using two different models: a 3D quarter and a 2D axisymmetric finite element model, due the geometry and loading conditions symmetry to reduce the computational time processing. The numerical simulations showed the shape deformation occurring after start the process and provided detailed quantitative information about expected weakness of the resulting piece.

Keywords: explicit dynamic, sheet metal, stamping.

INTRODUCTION

There are many industrial enterprises that use forming processes like deep drawing and stamping in order to produce sheet metal components with high productivity in large scale. In metal forming, a piece of material is plastically deformed to obtain the desired product with applications in different industrial areas (automobiles, trucks, airplanes, railway cars, locomotives, construction equipment, office furniture, office equipment...). Metal forming is one of the most important steps in manufacturing (Swadesh et al, 2014). In order to plastically deform a metal a force must be applied that will exceed the yield strength of the material. There are two different main classes of metal forming: the bulk and the sheet metal forming. Bulk deformation process is characteristic in that the work formed has a low surface area to volume ratio. The four basic bulk deformation processes are: rolling, forging, extrusion, wire and bar drawing. In sheet metal working, the metal being processed will have a high surface area to volume ratio. In this process cutting and forming operations are performed on relatively thin sheets of metal (0.4mm to 6mm). The three major categories are cutting, bending and drawing. The main advantages of the sheet metal parts are: high strength, good dimensional accuracy, good surface finish, relatively low cost for large quantities.

Until few years ago design of sheet metal forming was based on knowledge through work experience and expensive trial and error process. Nowadays the use of numerical simulations in different phases of the sheet metal forming process are performed using finite element analysis (Ranganath et al, 2012). Different codes are available for finite element analysis in metal forming such as Abaqus, Dynaform, Nike 2D, Ansys, etc. Also axisymmetric

deformation of sheet metal blanks is used to represent a cylindrical piece made of flat metal sheets, in pieces with axial symmetry. Different works demonstrated the relevance of this type of studies to prevent the occurrence of geometrical defects such as springback, wrinkling and surface appearance problems and to optimize various process variables, (Swadesh et al, 2014). Deep drawing is the metalworking process used for stamping flat sheets into cup-shaped forms, where the metal is subjected to different types of deformations (Dieter, 1961).

According to the main objective to obtain a successful stamping component, the engineer methodologies carry out the computer aided analysis based on the finite element method, (Firat, 2007). This tool helps the stamping methods engineer to reduce the costly and error iterations of the sheet metal forming with stamping criteria, (Firat, 2007).

In this work, in order to assess the stamping performance of the cylindrical sheet metal piece, different numerical simulations were performed in the finite element code, starting from a Benchmark problem. In all simulations the complete die-face design composed of the punch, binder and the die geometry is assumed to be rigid components. The blank material is the high strength steel of thickness 1mm. To increase the accuracy in the numerical simulations 3 different types of analysis were produced using: 3D quarter model with Solid 164 and Shell 163 elements; 3D quarter model only with Solid 164 elements and 2D axisymmetric model with Plane 162 element. This program is based on the explicit dynamic finite element (FE) method and incorporates the dynamic characteristics involved in this process.

STUDY CASE

Drawing is a sheet metal forming operation used to make cup shaped, box shaped, or other parts, (Groover, 2010). Figure 1 represents the basic drawing operation with the main dimensions and parameters.

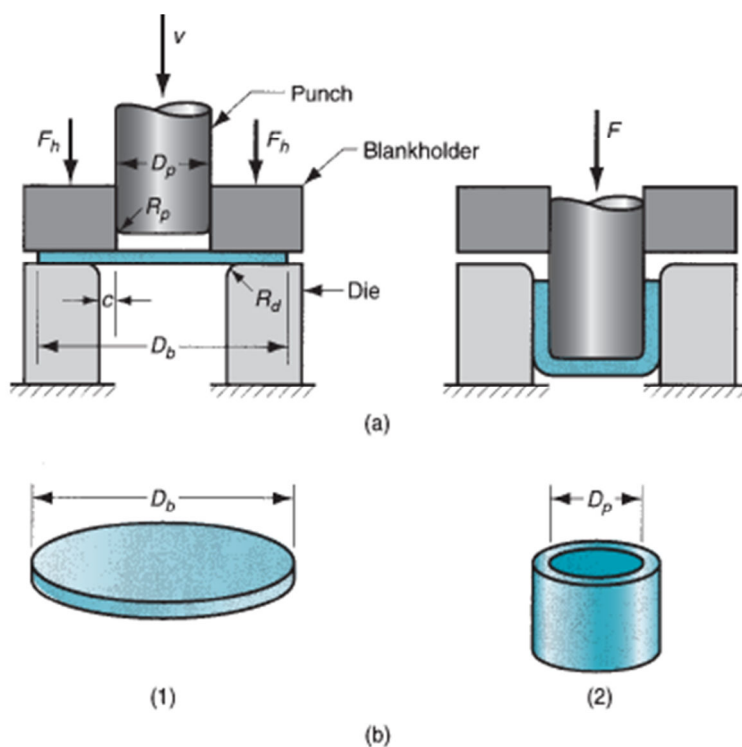


Fig. 1 - (a) Drawing of a cup shaped part: (1) start of operation, (2) near end of stroke. (b) corresponding workpart, (Groover, 2010).

There are various process parameters used in deep drawing process: punch force, drawing force, holding force, sheet metal blank thickness, punch velocity, punch stroke, coefficient of friction...

The most relevant dimensions considered in the present study of the punch (A), blank holder (B), sheet metal blank(C) and die (D) are shown in table 1, (Evangelista, 2000). The blank then has the same weight as the cup. A blank of correct thickness would be cut to the diameter necessary to produce this weight (Gyadari, 2013). Several formulas are available for calculating the blank diameter, as follow:

$$D_b = (D_p^2 + 4 D_p h)^{1/2} \quad (1)$$

Table 1 - Basic geometrical parameters

Parameters	Dimensions in mm
Punch diameter, D_p	50
Punch nose radius, $R_p = 4 t$ (as a start)	4
Punch thickness	4
Cup height of the first draw	25
Cup total height, h	29
Clearance between punch radius and die radius, $C \pm \geq 1,1 t$	1,5
Sheet metal blank diameter, D_b	100
Sheet metal blank thickness, t	1
Die profile radius	26,5
Die shoulder radius, R_d ($4 t \leq R_d \leq 8 t$)	8
Die cavity height	29
Blank holder radius	36,5
Blank holder thickness	2
Measures of drawing	
Drawing ratio, $DR=D_b/D_p \ll 2$ (only one stage)	2
Reduction, $r=(D_b - D_p)/D_b \ll 0,5$	0,5
Thickness to diameter ratio, $100 \times (t/D_b) \gg 1\%$	1%

Considered measures of drawing are necessary to verify, as calculated in table 1. According (Groover, 2010) the following limits need to be verified: the drawing ratio provides the severity of the operation according greater value; an approximate upper limit is a value of 2; the reduction value should be less than 0,5; the thickness of the starting blank to the blank diameter ratio is desirable to be greater than 1%, when decreases tendency for wrinkling increases. In cases where these limits on drawing ratios are exceeded, the blank must be drawn in two or more steps. For die design the recommended corner radius is $4t$ and increase it if necessary, the smaller the die radius the grater the force needed to draw the cup. The punch nose radius needs to be larger than needed and reduce its size in subsequent operations.

Carbon steel plate material SAE-AISI 1008 for sheet metal blank stamping is used. The mechanical properties are summarized in table 2, where σ_y is the yield stress, σ_u is the ultimate tensile strength and K the strength coefficient.

Table 2 - Material properties for steel alloy (AISI 1008)

Properties	Sheet metal blank
Density (kg/m ³)	7850
Young's Modulus (GPa)	210
Poisson's ratio	0,3
σ_y (MPa)	180 (180 to 310)
σ_u (MPa)	330 (330 to 370)
K (MPa)	660

From an analysis of the forces in equilibrium during the formation of a deep-drawn cup, the use of an approximate equation for the total punch force as a function of the diameter of the blank at any stage in the process. An approximate equation of the maximum drawing force F (in N) or punch load has been developed (Schey, 1987), (Groover, 2010), (Tschachtsch, 2005), (Kumar, 2016):

$$F = \pi D_p t \sigma_u (D_b/D_p - C) \quad (2)$$

C is a constant between 0,6 to 0,7, (Chaudhari,2016), (Kumar, 2016). The drawing force F varies throughout the downward movement of the punch, usually reaching its maximum value at about one-third the length of the punch stroke (Groover, 2010). A factor of safety should be taken. In the study case the draw force was equal to 80,8kN (=67,4kN x 1,2).

A holding force was applied directly on the blank holder, usually represents 33% of the drawing force ($F_h=33\% \times 80,8= 26\text{kN}$), (Kumar, 2016). According the dimensions and the material, the calculated holding force was equal to 26kN. The holding force F_h (in N) is an important factor in drawing operation.

In the case of low blank holder force there is wrinkle usually in the flange of the drawn part, when increasing wrinkle is reducing. But large value of the blank holder force will cause fracture in the material. The press force capacity can be calculated by using the adding between the draw force using a safety factor and the blank holding force, (Chaudhari,2016). In the study case the value is equal 108kN (or 11 ton). Other considerations should be considered in deep drawing to control the sheet metal fracture, as excessive friction between blank and punch, insufficient clearance between punch and die and insufficient punch or die corner radius.

EXPLICIT DYNAMIC FEM MODEL

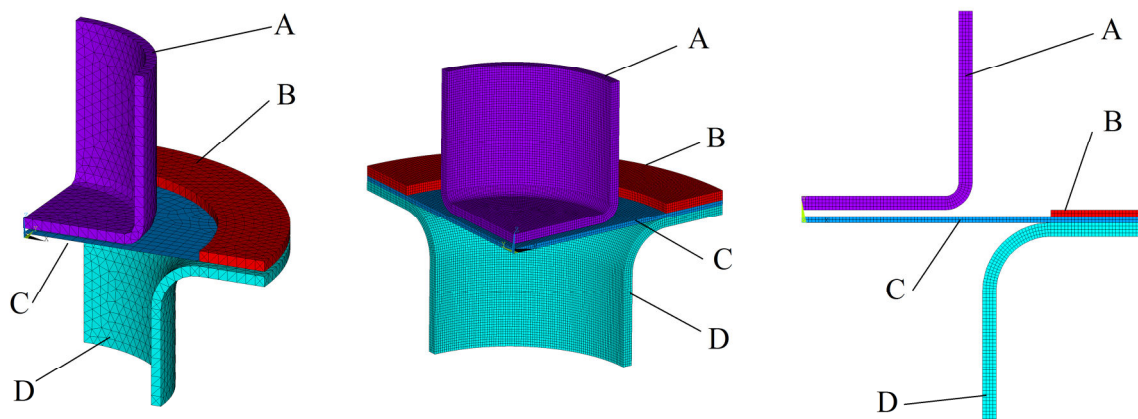
An explicit dynamic finite element (FEM) model was developed using ANSYS program, most helpful in solving high deformation time dependent problem. The program is a nonlinear transient dynamic finite element method with both explicit and implicit solvers. ANSYS is widely used in applications of sheet metal forming (as metal stamping, hydro forming,

forging, deep drawing, multi-scale processes) predicting the level of stresses and deformations in the metallic material and determine if the metal will fail, (Abdulla, 2013).

Explicit scheme is a function of time, where the velocity and acceleration as well as the mass and damping need to be considered in this scheme. Central difference time integration is used to calculate field variables at respective nodal points, particularly suited for nonlinear problems. The equation of motion is evaluated at the previous time step and works in time step increments, where the displacements are calculated as the time proceeds. Gradually as time would progress the deformation also would change. The time step for implicit solvers is about two to three orders of magnitude of the explicit time step. The explicit solvers are best suited to treat problems of short duration, with high loading velocity and highly nonlinear nature that require small time steps for accuracy reasons.

The proposed numerical model intends to simulate the stamping process, as a practical problem with industry interest, and to evaluate the mechanical stress and deformation in stamping piece. The input die assembly model (die, blank, blank holder and punch) were constructed in pre-processor using tool CAD-geometry in the FEM program. After the generated CAD model, a fine meshing is created.

Figure 2 represents different meshes used in this work. In all simulations the complete die-face design composed of the punch (A), binder or blank holder (B) and the die geometry (D) is assumed to be rigid components. The sheet metal blank material (C) is the high strength steel of thickness 1mm.



(I): 3D quarter: Solid164 (8nodes) / Shell163 (4nodes) (II): 3D quarter: Solid164 (8nodes) (III): 2D Axisymmetric: Plane162 (4nodes)

Fig. 2 - Different meshes of die assembly,
(A-Punch, B-Binder or Blank Holder, C-Sheet metal blank, D-Die).

To increase the accuracy in the numerical simulations 3 different types of analysis were produced using: (I) three dimensional (3D) quarter model with Solid 164 and Shell 163 elements; (II) three dimensional (3D) quarter model only with Solid 164 elements and (III) two dimensional (2D) axisymmetric model with Plane 162 element. A discretized mesh was used, according a length of the finite element equal to 1mm (I) and 0,5mm (I and III), as represented in figure 1. Also, the definition of the materials and tool motions are important steps in forming simulation.

All boundary conditions will be considered due the geometry and dynamics characteristics involved in the process. In 3D problem only a quarter of the geometry was modelled and symmetry boundary conditions used. In 2D problem due to axisymmetric boundary conditions the central left region nodes will not move in the left direction due to the constraint.

For dynamic process, punch is displayed vertically downwards to establish contact with the blank with a punch speed equal to 1,7m/s. To improve the material flow control, a constant blank holder force is used, equal to $F_h=26\text{kN}$ and applied vertically in binder to simulate a contact between the blank holder and the blank.

In ANSYS contact interaction between all components in die assembly was formulated using the `*CONTACT_AUTOMATIC_SURFACE_TO_SURFACE`, generally recommended for this type of process. Thus, by choosing automatic surface to surface contact, the program will automatically adjust for the changes which occur during this simulation.

Contact assuming Coulomb friction between the blank, punch, die surface and the blank holder was modelled. In ANSYS its behaviour can be controlled with the parameters on the `*CONTACT_...` card: static and dynamic friction coefficients, exponential decay coefficient and coefficient for viscous friction. It is required the definition of the frictional coefficients. Dynamic friction coefficient between the blank and the die surface was very low (friction coefficient 0,04 or lower) due to the lubricant (Yasar and Kadi, 2007). The dynamic friction coefficient between the blank and the blank holder and the punch was assumed with the same value equal to 0,04. Static friction coefficient between all components for cold working was equal to 0,1 (Groover, 2010).

Also, time step size is automatically adjusted to satisfy the contact between different bodies and the program checks all elements when calculating the required time step. In explicit analysis, the time step is affected by element size and material sound speed. For stability reasons, a scale factor (0,9) is used to decrease the time step. In all simulations the time step used was equal to 0,00015s until end time 0,015s, estimate time due the cup height and the punch speed.

To define the material model of the metal should be taken in consideration the strain-rate with dependency of the isotropic material plastic curve. The rate sensitive power law plasticity model is a strain rate dependent plasticity model typically used for superplastic forming analysis. The sheet metal blank is made of a steel alloy material that is assumed to satisfy the stress-strain curve Ramburgh-Osgood as constitutive relationship, (Groover, 2010):

$$\sigma = K \varepsilon^m \dot{\varepsilon}^n \quad (3)$$

where σ is the stress; K is the reference stress value or strength coefficient; m the hardening coefficient equal to 0,012; n is the strain rate sensitivity coefficient considered 0,19; ε the true strain and $\dot{\varepsilon}$ the initial strain rate considered equal to 30, (Evangelista, 2000). All these parameters permit to identify the correspondent plastic flow curve.

In the numerical model, the Ramburgh-Osgood law is applied through the `*MAT_RATE_SENSITIVE_POWERLAW_PLASTICITY` (`*MAT` type 64) in ANSYS material library. The punch, binder and die were modelled as a rigid body in order to reduce the computing time and resources.

NUMERICAL RESULTS

The main objective is the numerical model forming a cylindrical cup of 50 mm diameter and 25 mm deep in 0,015s. Deformed shapes of the blank at different drawing depth are presented in figure 3, for each numerical model. Models (I) (II) and (III) were simulated with a complete die assembly including the blank holder and the effect of the applied holding force. Also, only the Model (I) was simulated without the blank holder part to verify the flange wrinkling effect during the sheet metal forming.

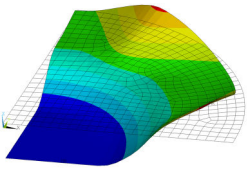
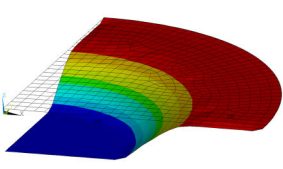
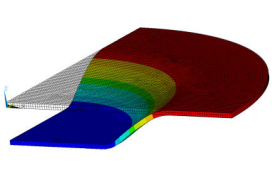
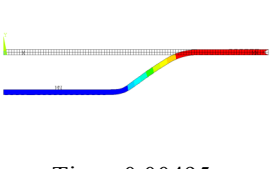
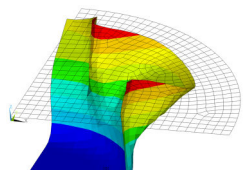
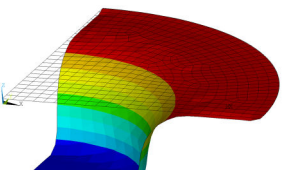
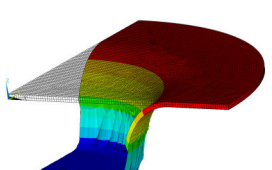
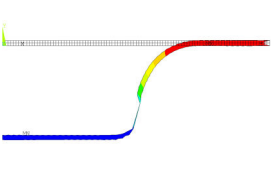
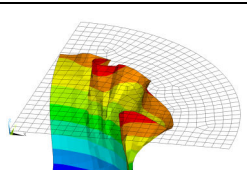
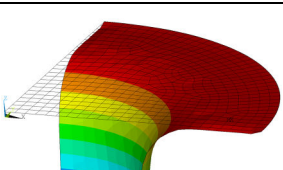
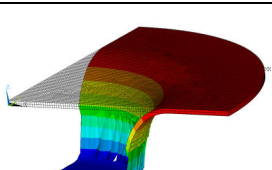
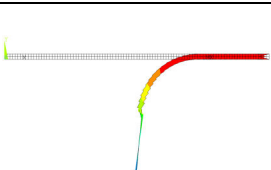
Die assembly without blank holder	Die assembly with blank holder and applied holding force		
Model (I)	Model (I)	Model (II)	Model (III)
 <p>Time=0,00495s Depth=0,00809m</p>	 <p>Time=0,00495s Depth=0,007982m</p>	 <p>Time=0,00495s Depth=0,007458m</p>	 <p>Time=0,00495s Depth=0,007445m</p>
 <p>Time=0,01095s Depth=0,018229m</p>	 <p>Time=0,01095s Depth=0,018069m</p>	 <p>Time=0,01095s Depth=0,017764m</p>	 <p>Time=0,01095s Depth=0,017697m</p>
 <p>Time=0,015s Depth=0,025127m</p>	 <p>Time=0,015s Depth=0,024939m</p>	 <p>Time=0,01155s Depth=0,018833m</p>	 <p>Time=0,015s Depth=0,024616m</p>

Fig. 3 - Different deformed shapes of the blank

Considering the results at different time instants and considering all die assembly, it is observed that the model (I) represents the total depth circular cup drawing. With model (II) only 18,8mm in depth is obtained. 2D Axisymmetric Model (III) permits to calculate the cup depth of 24,6mm, however the deformed thickness sheet metal decrease significantly.

In all meshes until time instant to obtain 18mm drawing depth the results are in good agreement, but increasing the time punch progression the results become no closer to a converging values.

Shell elements in model (I) allow the modelling of thin blank with fewer elements than solid elements, easier to mesh and less prone negative Jacobian errors, which might occur when using thin solid features. 3D solids elements are best for thick combinations.

Model (I) without the blank holder intends to demonstrate the instability effect in the body of the piece with a deep drawing simulation. It is necessary blank holder load to prevent wrinkling. Increasing this force by using a blank holder of an appropriate size will ensure a reduction in wrinkling height, (Patel, 2014).

Figure 4 represents the distribution of equivalent plastic strains for successive trails of the stamp falling in time instants in expanded model. Maximum plastic deformations are observed with red colour region or in grey colour with higher values than model (I). The blue colour represents minimum strains in the sheet metal blank. In this study, generally maximum strain occurs on the lateral wall cup where tension in side wall is higher.

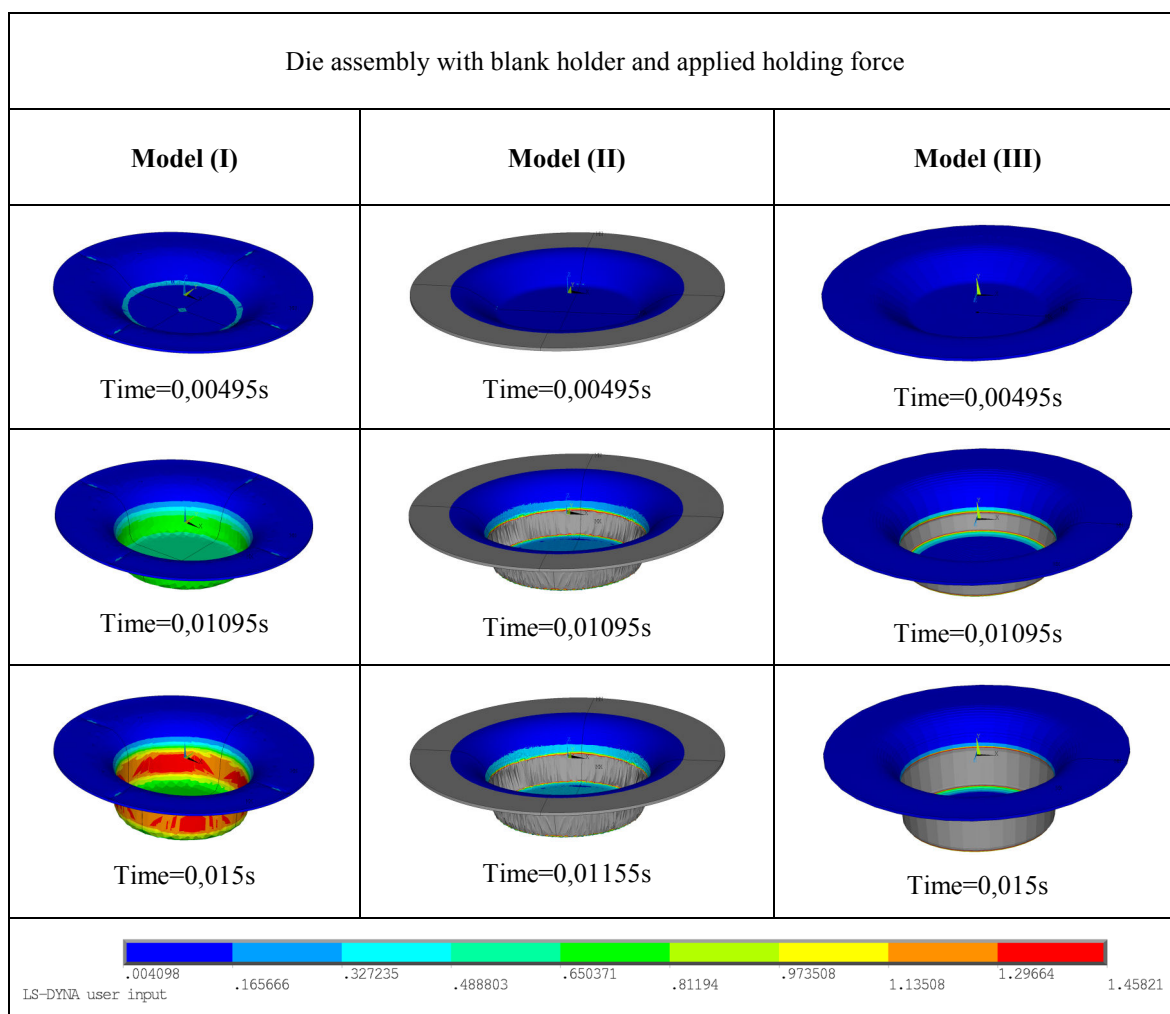


Fig. 4 - Equivalent plastic strain in expanded deformed shapes of the blank

CONCLUSION

The results from the numerical simulations produced comparisons between all types of finite element models. And it was observed that there was a good correlation between all different simulations until time instant to obtain a cup of 18mm drawing depth.

This study shows the finite element procedures for the sheet metal forming process and presents the performance of the use of axisymmetric conditions in the balance of the computational requirements against the desired accuracy of the results.

With numerical simulations it is possible to observe the quality of the piece according the thickness distribution, and also detects some defects like wrinkling, crushing and tearing material.

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