

Optimization of a Fuzzy Logic Controller for MR Dampers Using an Adaptive Neuro-Fuzzy Procedure

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Received 12 March 2016

Accepted 21 August 2016

Published 9 December 2016

Intelligent and adaptive control systems are naturally suitable to deal with dynamic uncertain systems with non-smooth nonlinearities; they constitute an important advantage over conventional control approaches. This control technology can be used to design powerful and robust controllers for complex vibration engineering problems such as vibration control of civil structures. Fuzzy logic based controllers are simple and robust systems that are rapidly becoming a viable alternative for classical controllers. Furthermore, new control devices such as magnetorheological (MR) dampers have been widely studied for structural control applications. In this paper, we design a semi-active fuzzy controller for MR dampers using an adaptive neuro-fuzzy inference system (ANFIS). The objective is to verify the effectiveness of a neuro-fuzzy controller in reducing the response of a building structure equipped with a MR damper operating in passive and semi-active control modes. The uncontrolled and controlled responses are compared to assess the performance of the fuzzy logic based controller.

Keywords: Structural control; MR damper; fuzzy logic control; ANFIS.

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1. Introduction

Soft computing techniques have been shown to be effective in dealing with complex and nonlinear behavior of structural control systems. Fuzzy logic controllers can closely imitate human reasoning and control procedures enabling the use of previous experience and experimental results in designing simple and model-free control systems. The main problem relies on the optimization of the fuzzy parameters, in particular the definition of a truthful inference system. In this regard, neuro-adaptive learning techniques such as ANFIS constitute simple methodologies to optimize fuzzy systems. This allows learning information about a dataset in order to compute the membership function parameters that best allow the associated fuzzy inference system to track a given input/output data. ANFIS is a hybrid learning algorithm that combines the backpropagation gradient descent and least squares methods to create a fuzzy inference system whose membership functions are iteratively adjusted according to a given set of input and output data. The reasoning scheme of ANFIS architecture and its inherent variables are shown in Fig. 1, see Refs. 1 and 2. The inherent advantages of these neuro-fuzzy systems make them particularly suitable to develop control systems for structural engineering problems, which typically have uncertain parameters and nonlinear behavior. Besides, fuzzy based controllers allow a model-free estimation of the system and the fuzzy controller can be developed by encoding the knowledge of the system without the need to state how the outputs depend mathematically upon the inputs.

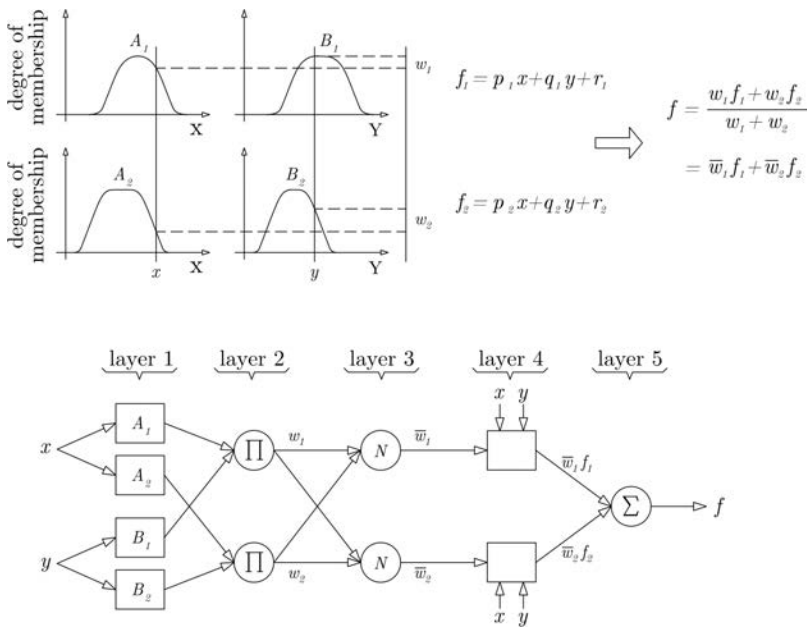


Fig. 1. The scheme of adaptive neuro-fuzzy inference system or ANFIS (see Refs. 1 and 2).

This paper describes the development of a fuzzy logic-based controller designed to reduce seismic-induced vibrations in a three degrees-of-freedom (3-DOFs) structural system representing a three-storey shear building structure by using a MR damper. These semi-active devices represent a promising technology with great potential for structural control applications. However, they present a highly nonlinear hysteretic behavior that needs to be considered in designing the control system. The design process of fuzzy controllers requires the characterization of several parameters (inputs, output, membership functions and the corresponding universe of discourse) and the definition of inference rules that relates the inputs to obtain the desired output. This last operation can be carried out using only previous human knowledge about the system behavior or optimization techniques. However, inference rules of complex and/or nonlinear systems cannot be conveniently modeled by simple human knowledge demanding the use of optimization tools to obtain the relationship between the inputs and output for a given problem.^{1,2} This problem is addressed in this study in which ANFIS is used to optimize a set of fuzzy rules to design a fuzzy based controller to mitigate the vibration of a structural system.^{3,4} The optimization procedure with ANFIS is discussed in the development of the controller. The semi-active control system is derived from the data obtained with an optimal controller, which is used as a reference to train a fuzzy based controller via ANFIS. The uncontrolled response is compared with passive and semi-active controlled responses in order to assess the performance of the proposed fuzzy controller. This paper is based upon Braz-César and Barros,⁵ but this previous paper has been enhanced including additional information about ANFIS model, a more detailed characterization of the response obtained with the passive control modes, the inclusion of new performance criteria to better evaluate the effectiveness of the proposed fuzzy controller, and also references have been extended and updated.

2. Numerical Model

An effective semi-active control system involves an appropriate control algorithm that can take advantage of the dissipative properties of the control device, i.e. the MR damper.^{6,7} There are several approaches available in the literature to control semi-active devices including soft computing techniques such as neuro-fuzzy controllers. Thus, a numerical analysis was carried out to evaluate the performance of a semi-active neuro-fuzzy controller in reducing the response of a simple structural system. The numerical model represents a three degrees-of-freedom system under seismic excitation as shown in Fig. 2. A control force generated by an actuator (such as a MR damper) located between the base and the first mass, can be applied in order to reduce the system response. The damping force can be adjusted using a control system that based upon the system response computes the required damping force that should be applied to the system changes the system response in order to improve and control the structural performance.

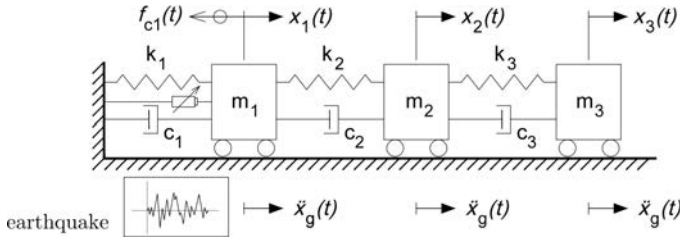


Fig. 2. Structural system controlled with a MR damper at the first floor.

The parameters of the 3-DOFs system are as follows: $m_1 = m_2 = m_3 = 100$ kg; $c_1 = 125$ N·s/m, $c_2 = c_3 = 50$ N·s/m; $k_1 = k_2 = k_3 = 6 \cdot 10^5$ N/m. According to the representation depicted in Fig. 2, the structural system can be described by means of the following matrices:

$$\mathbf{M} = \begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix}, \quad (2.1)$$

$$\mathbf{C} = \begin{bmatrix} c_1 + c_2 & -c_2 & 0 \\ -c_2 & c_2 + c_3 & -c_3 \\ 0 & -c_3 & c_3 \end{bmatrix}, \quad (2.2)$$

$$\mathbf{K} = \begin{bmatrix} k_1 + k_2 & -k_2 & 0 \\ -k_2 & k_2 + k_3 & -k_3 \\ 0 & -k_3 & k_3 \end{bmatrix}, \quad (2.3)$$

where \mathbf{M} is the mass matrix, \mathbf{C} is the damping matrix and \mathbf{K} is the stiffness matrix of the system. The natural frequencies and the corresponding mode shapes (inside brackets) are: $freq_1 = 5.48$ Hz $(-0.328, -0.591, -0.737)$, $freq_2 = 15.37$ Hz $(-0.737, 0.328, 0.591)$ and $freq_3 = 22.21$ Hz $(-0.591, 0.737, -0.328)$. The equations of motion of the controlled structure can be defined by a state space formulation given by

$$\dot{\mathbf{z}}(t) = \mathbf{A}\mathbf{z}(t) + \mathbf{B}f(t) + \mathbf{E}\ddot{x}_g(t), \quad (2.4)$$

where $\mathbf{z}(t) = [\mathbf{x}(t), \dot{\mathbf{x}}(t)]$ is the state vector, \mathbf{A} is the system matrix defined by

$$\mathbf{A} = \begin{bmatrix} \mathbf{0} & \mathbf{I} \\ -\mathbf{M}^{-1}\mathbf{K} & -\mathbf{M}^{-1}\mathbf{C} \end{bmatrix}. \quad (2.5)$$

\mathbf{B} is an additional matrix accounting for the position of the control forces in the structure, $f(t)$ is the control force, \mathbf{E} is the disturbance vector and $\ddot{x}_g(t)$ represents the seismic excitation loading. In this case there is only one control force applied to the first mass and then

$$\mathbf{B} = [\mathbf{0} \quad \mathbf{M}^{-1}\mathbf{\Gamma}]^T, \quad (2.6)$$

where $\mathbf{\Gamma} = [1, 0, 0]$ is the location vector. The earthquake excitation influence vector is defined by

$$\mathbf{E} = [\mathbf{0} \quad -\mathbf{\Lambda}]^T \quad (2.7)$$

in which $\mathbf{\Lambda} = [1, 1, 1]$ represents the location of the external loading. The system response can be determined using the state space output vector defined as

$$\hat{\mathbf{y}}(t) = \hat{\mathbf{C}}\mathbf{z}(t) + \hat{\mathbf{D}}f(t) + \hat{\mathbf{F}}\ddot{x}_g(t). \quad (2.8)$$

If the system displacements, velocities and accelerations are required, then

$$\hat{\mathbf{C}} = \begin{bmatrix} \mathbf{I} & \mathbf{0} \\ \mathbf{0} & \mathbf{I} \\ -\mathbf{M}^{-1}\mathbf{K} & -\mathbf{M}^{-1}\mathbf{C} \end{bmatrix}, \quad (2.9)$$

$$\hat{\mathbf{D}} = [\mathbf{0} \quad -\gamma]^T, \quad (2.10)$$

$$\hat{\mathbf{F}} = [\mathbf{0} \quad -\mathbf{\lambda}]^T, \quad (2.11)$$

where $\hat{\mathbf{C}}$ is the output matrix, $\hat{\mathbf{D}}$ is the feedthrough vector related with the control force and $\hat{\mathbf{F}}$ is the disturbance signal location vector. In this case, $\gamma = [1, 0, 0]$ and $\mathbf{\lambda} = [1, 1, 1]$ represent the location of the control force and of the earthquake acceleration excitations at each floor level, respectively.

The N-S component of the El-Centro ground motion (1940, peak acceleration of 3.42 m/s^2) will be used as the excitation signal. The system represents a small-scale building structure and the earthquake signal was also scaled to characterize the magnitude of displacements that would be observed in experiments tests. In this case, the time was scaled to 20% of the full-scale earthquake time history as displayed in Fig. 3.

The response of the structure can be controlled using a MR damper located between the ground and the first floor. A small-scale MR damper based on the RD-1005-03 prototype (Lord Corp., USA) is used in this study. The device has a

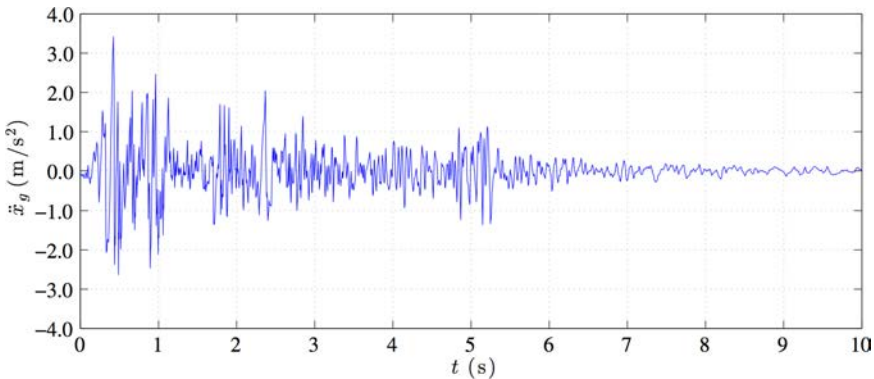


Fig. 3. Time-scaled El-Centro NS earthquake excitation (0.2t).

conventional cylindrical configuration with 41.4 mm in diameter, 208 mm long in its extended position allowing a stroke of ± 25 mm. The device can operate within a current range from 0.0 A up to 2.0 A with a usual value of 1.0 A for continuous operation. This MR damper can generate a peak force of 2224 N (1.0 A) using a small battery with less than 10 W. The device will be numerically modelled using the so-called modified Bouc–Wen model,^{8,9} which is a usual phenomenological model to simulate the hysteretic behavior of this type of semi-active actuators. The mechanical configuration of the modified Bouc–Wen model is shown in Fig. 4.

Thus, the predicted damper force F_{MR} can be obtained from the schematic representation of the modified Bouc–Wen model (see Ref. 8) depicted in Fig. 4 by

$$F_{MR} = c'_1 \dot{y} + k'_1(x - x_0), \quad (2.12)$$

in which

$$\dot{y} = \frac{1}{(c'_0 + c'_1)} [\alpha s + c'_0 \dot{x} + k'_0(x - y)] \quad (2.13)$$

is dependent on the evolutionary variable

$$\dot{s} = -\beta |x|s|x|^{(n-1)} - \zeta \dot{x}s^n + \delta \dot{x}. \quad (2.14)$$

Generally, α , c'_0 and c'_1 are current dependent parameters described by

$$\alpha(i) = \alpha_a + \alpha_b i, \quad (2.15)$$

$$c'_0(i) = c'_{0a} + c'_{0b} i, \quad (2.16)$$

$$c'_1(i) = c'_{1a} + c'_{1b} i, \quad (2.17)$$

where i is the operating current. The parameters of the modified Bouc–Wen model are given in Table 1.⁹ The dynamics involved in the MR fluid reaching equilibrium state is represented through first order filter given by

$$\dot{u} = \eta(u - i) \quad (2.18)$$

in which the applied current u is described with a time delay relative to the desired current i and η is the filter constant (in this case, $\eta = 130 \text{ s}^{-1}$).

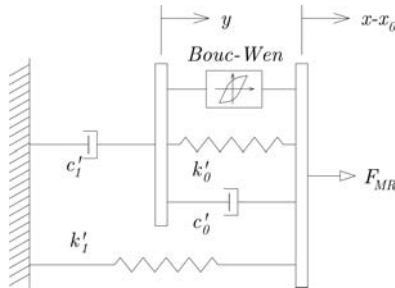


Fig. 4. Phenomenological model of the MR damper (modified Bouc–Wen model).

Table 1. Modified Bouc–Wen model parameters (RD-1005-3 MR damper) (see Ref. 9).

Parameters	δ [-]	β [mm ⁻¹]	ζ [mm ⁻¹]	k'_0 [N/mm]	f_0 [N]	n
Constant	10.013	3.044	0.103	1.121	40	2
Current dependent	$\alpha(i) = -826.67i^3 + 905.14i^2 + 412.52i + 38.24$ [N]					
	$c'_0(i) = -11.73i^3 + 10.51i^2 + 11.02i + 0.59$ [N·s/mm]					
	$c'_1(i) = -54.40i^3 + 57.03i^2 + 64.57i + 4.73$ [N·s/mm]					

Using the state space formulation, the uncontrolled response of the 3-DOFs system under the earthquake ground motion is displayed in Fig. 5. It should be noted that the response was obtained with a high excitation level of the El Centro earthquake achieved by scaling up the amplitude of the earthquake signal in 150%.

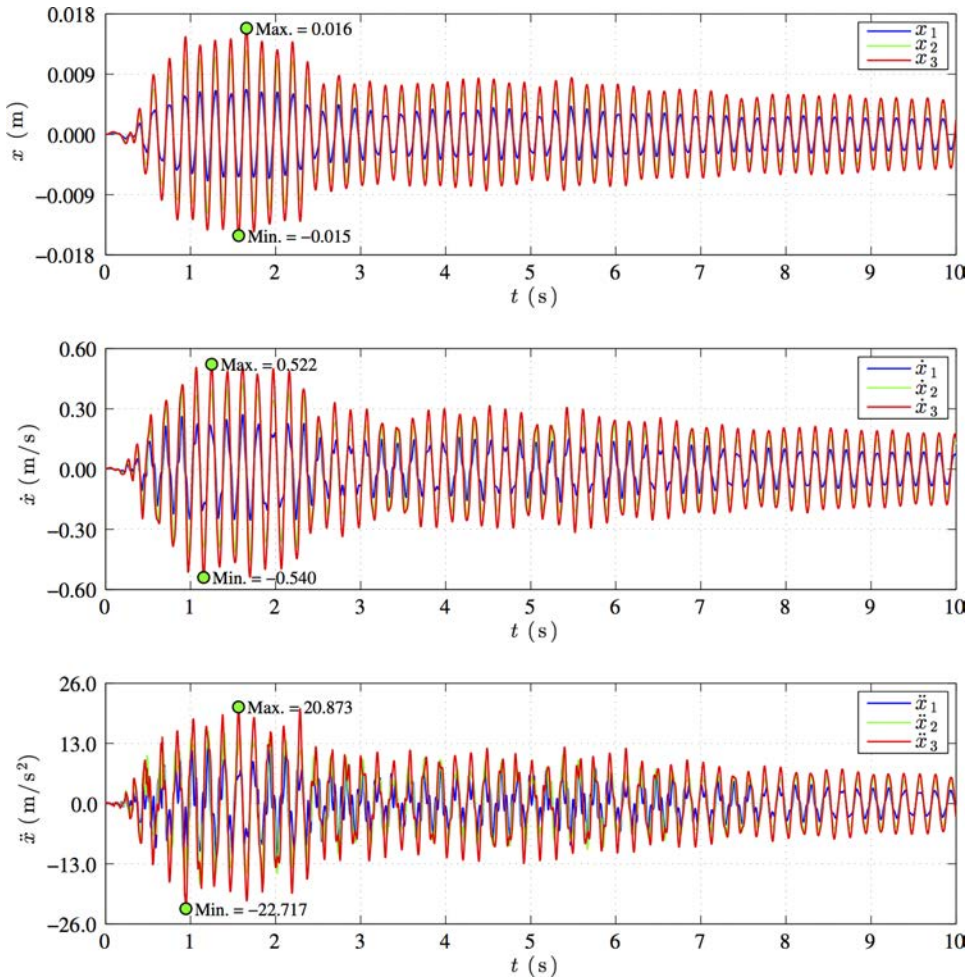


Fig. 5. Uncontrolled responses of the 3-DOFs system.

3. Optimization of a Fuzzy Based Controller Using ANFIS

The fuzzy logic based controller was designed using ANFIS to find the nonlinear map that best fits the expected response of the control system. The development of a neuro-fuzzy based controller typically involves four main steps:

- (i) Definition of input variables and the corresponding fuzzy inference system (FIS) membership functions (the FIS output is the desired control signal);
- (ii) Selection of data sets to generate training and checking data;
- (iii) Use of ANFIS optimization algorithm for training the FIS membership function parameters to model the set of input/output data by mapping the relationship between inputs and outputs in order to generate a fuzzy model of the systems;
- (iv) Validation of the resulting fuzzy model.

ANFIS training procedure is summarized in the flowchart shown in Fig. 6. The process begins by obtaining a training data set and checking data sets. The training data is used to find the premise parameters for the membership functions (MFs are dependent on the system designer). A threshold value for the error between the actual and desired output is determined. The consequent parameters are found using the least-squares method. If this error is larger than the threshold value, then the premise parameters are updated using the gradient decent method. The process ends when the error becomes less than the threshold value. Checking data set can then be used to compare the model with the actual system.

The data sets for training and validation can be obtained from numerical and experimental data. Usually, numerical training data provide better results in optimizing the fuzzy inference system since experimental results can have sparse or incomplete making the optimization procedure more difficult. In this case, the

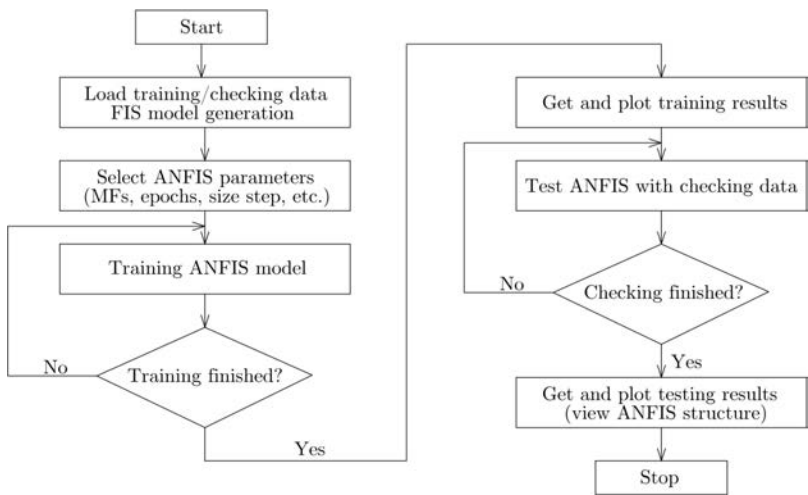


Fig. 6. Flowchart of ANFIS training.

neuro-fuzzy controller was developed based on the numerical results of a linear quadratic Gaussian (LQG) controller whose response is used to define the training data set for the neuro-fuzzy optimization procedure with ANFIS. Floors accelerations and the displacement across the MR damper are the responses of the controlled system used by the LQG controller to determine the desired control force. The control signal is determined from the predicted control force using an inverse Bingham model of the MR damper. The system responses and the desired control signal were recorded and then used to train the neuro-fuzzy controller. Thus, the data sets for training and validation were obtained from a LQG controlled system to a set of amplitude-scaled versions of the El Centro NS earthquake excitation (i.e. 100 gal, 200 gal, 335 gal and 500 gal seismic accelerations). The LQG controller combines a linear quadratic regulator (LQR) algorithm with a Kalman filter estimator. Identically distributed Gaussian white noise is used to simulate acceleration noise measurements. Regarding the LQR controller, the state gain matrix \mathbf{G} is tuned through the weighting matrices \mathbf{Q} and \mathbf{R} . In the present example different configurations of these parameters were evaluated by measuring the effect of each combination in the system response. The following weighting parameters provided the best performance in reducing in the structural response

$$\mathbf{Q} = \begin{bmatrix} \mathbf{K} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} \end{bmatrix}, \quad (3.1)$$

$$\mathbf{R} = r = 5e^{-7}. \quad (3.2)$$

The observer gain \mathbf{L} must be adjusted to achieve the required performance. A high gain allows the filter to follow the observations more closely while a low gain follows the predictions more closely. This is accomplished by setting

$$\mathbf{Q}_\omega = q_\omega \mathbf{I}_e, \quad (3.3)$$

$$\mathbf{R}_v = r_v \mathbf{I}_m, \quad (3.4)$$

where q_ω and r_v are weighing factors, and \mathbf{I}_e and \mathbf{I}_m are identity matrices related with the number of excitation inputs and measurement signals, respectively. A common approach is to set one of the tuning parameters and adjust the other parameter until the result is satisfying. In this case $\mathbf{I}_e = 1(\ddot{x}_g)$ and $\mathbf{I}_m = \mathbf{I}_{4 \times 4}(x, \dot{x}_1, \ddot{x}_2, \ddot{x}_3)$.

The recorded velocity and the control signal from the LQG controller were used to define the training data for the fuzzy controller. The first and third floor velocities are the FIS inputs while the current represents the fuzzy output. The choice of the first floor velocity as input variable occurs since the MR damper was placed there on a non-collocated configuration. It is known that in general, viscous type damping is dependent on relative velocity. The third floor velocity was chosen as input variable because of being the one that, according to the vibration mode shapes, will have the highest influence on the system response. Such two input variables were chosen to represent the dependence of the input current, because of the easiness of human visualization and interpretation of the controlled output signal through the use of

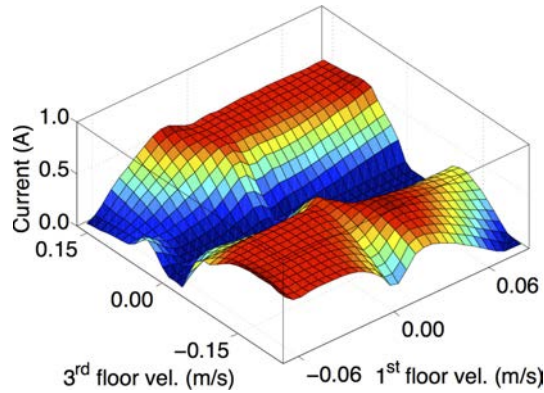


Fig. 7. Fuzzy surface of the neuro-fuzzy controller.

three-dimensional surfaces. This obviously simplifies the proposed fuzzy controller approach.

An initial, increasing and decreasing step sizes of 0.12, 1.20 and 0.8, respectively during 200 epochs are the optimization parameters. The optimal number of membership functions (MFs) was defined through a trial-and-error process. In this case, six bell-shaped MFs were used to model each input variable (first and third floor velocities). The resultant fuzzy surface is shown in Fig. 7.

When the first and third floor velocities are large and have the same signs, the required control signal is also large. When both velocities are large but have opposite signs, the fuzzy controller delivers the lowest control signal. Besides, the minimum damping force requirement is located around the central zone comprising small floor velocities.

4. Numerical Results

The MR damper can operate in two modes: as a passive energy dissipation device, i.e. without a control system (the properties of the actuator are constant during the simulation) and as a semi-active actuator whose control action is being commanded by a neuro-fuzzy based controller. A set of numerical simulations was carried out to obtain the response of the three DOF structure using the MR damper in a passive OFF mode (zero voltage/current input), passive ON mode (maximum value of the operating current) and semi-active control mode. The results of the a neuro-fuzzy semi-active control system are compared with the uncontrolled, passive OFF and passive ON responses to evaluate the efficiency of the semi-active control scheme in reducing the structural response. A Simulink model of the semi-active control system is shown in Fig. 8.

A new numerical simulation was carried out to obtain the responses of the structure for the passive OFF and passive ON modes with the MR damper

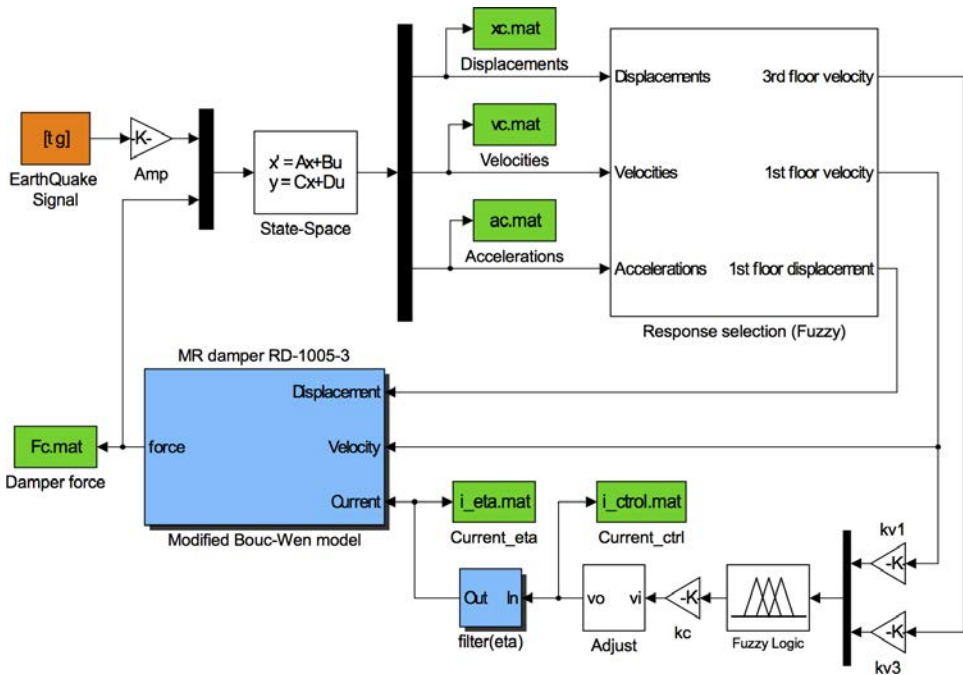


Fig. 8. Simulink model of the semi-active control system.

represented by the modified Bouc–Wen model. The system response for the passive OFF case along with the uncontrolled response of the 3rd floor is shown in Fig. 9.

The corresponding force-velocity and force-displacement responses, i.e. damping force plots for the passive OFF configuration (passive energy dissipation with $i = 0.00$ A) are shown in Fig. 10. Likewise, the system response achieved with the passive ON configuration (passive energy dissipation with $i = 0.50$ A) along with the uncontrolled response is shown in Fig. 11. The corresponding force-velocity and force-displacement responses for this control mode is shown in Fig. 12.

The results show a considerable vibration reduction using the MR damper as a passive energy dissipation device in a passive ON configuration. The increase of the operating current in the MR damper has a significant effect in the damping force and the energy dissipation capacity of the device. Figure 13 displays the structural response of each floor obtained with the proposed fuzzy based control system along with the uncontrolled response of the third floor during the numerical simulation. As can be seen, the proposed semi-active control system achieves a good performance in reducing the structural responses using only floor velocities as the reference (input) signals to compute the control action. In fact, the main advantage of this fuzzy logic based control system is that only the first and third floor velocities of the structure are required to determine the desired control signal. This means that the damping

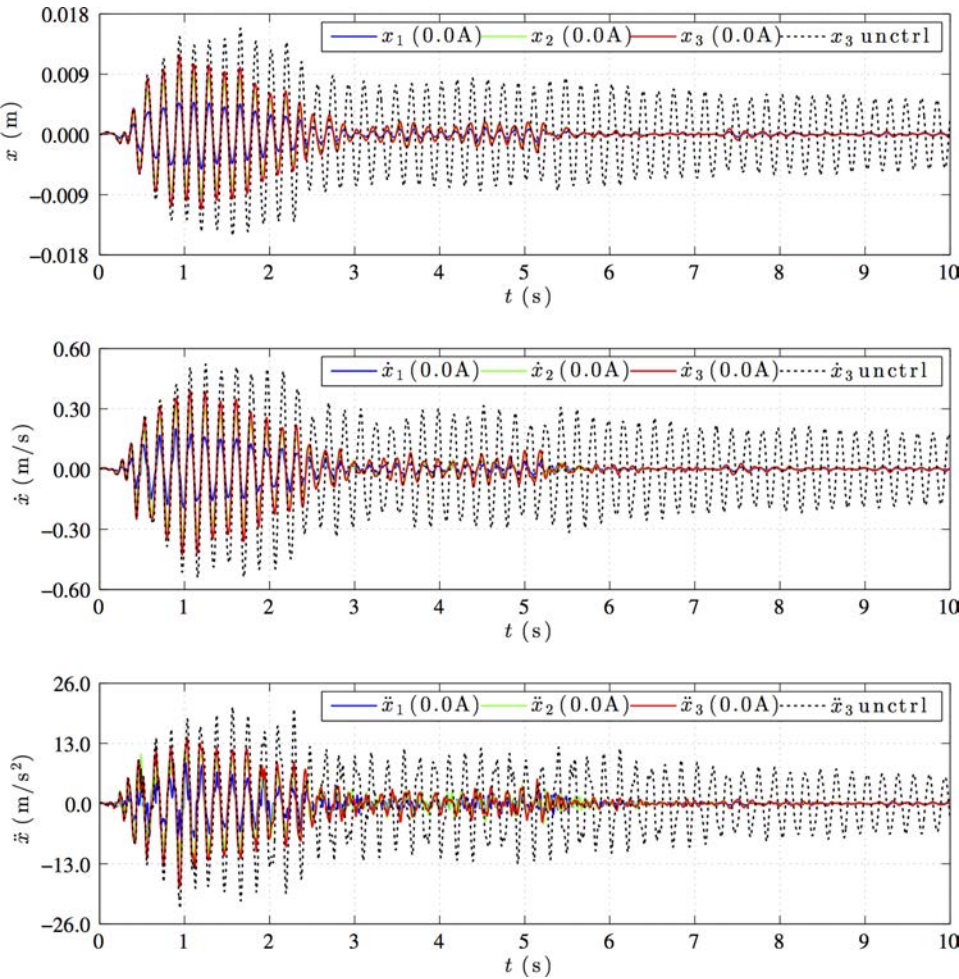


Fig. 9. Structural response with a passive OFF control mode.

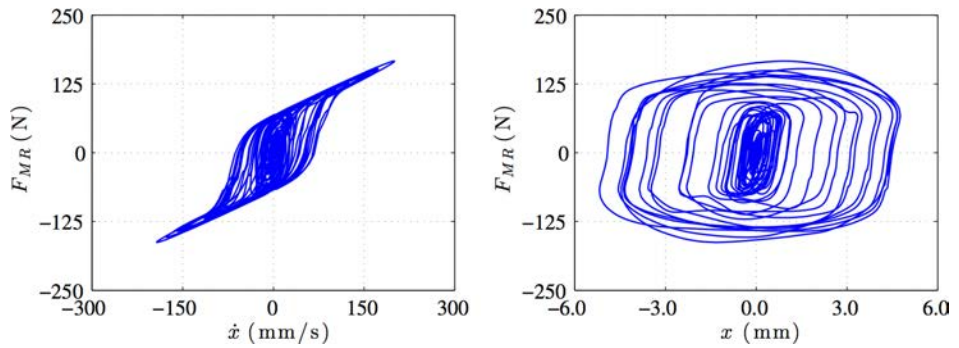


Fig. 10. MR damper force in a passive OFF control configuration.

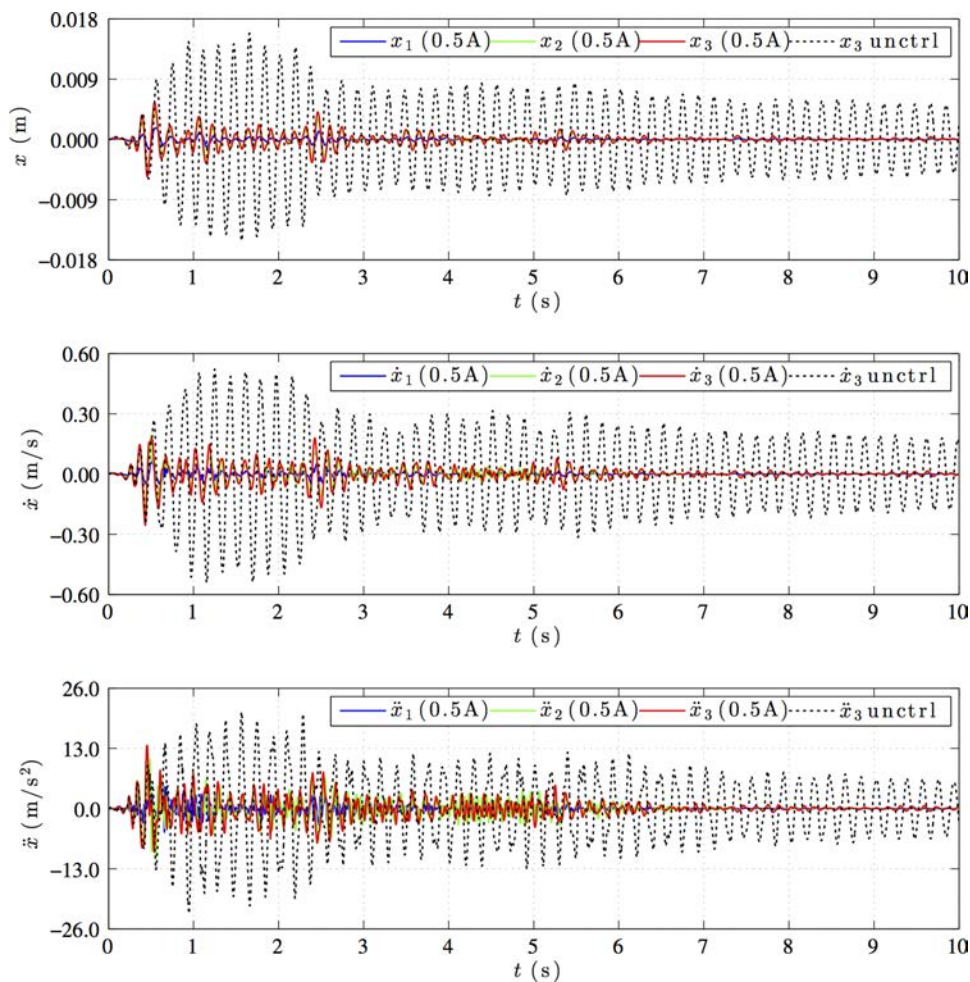


Fig. 11. Structural response with a passive ON control mode.

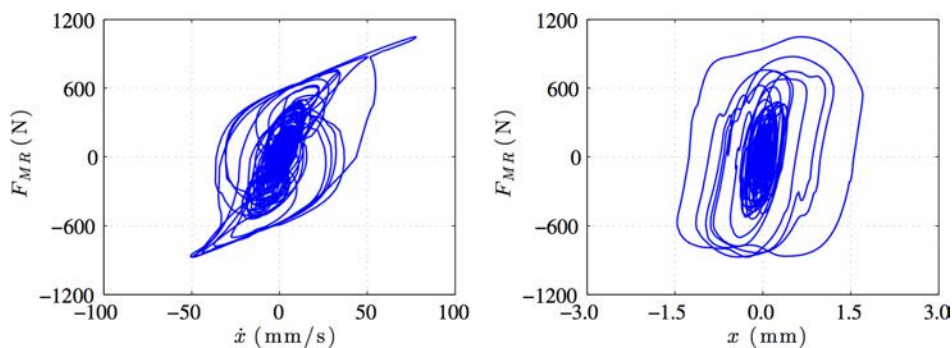


Fig. 12. MR damper force in a passive ON control configuration.

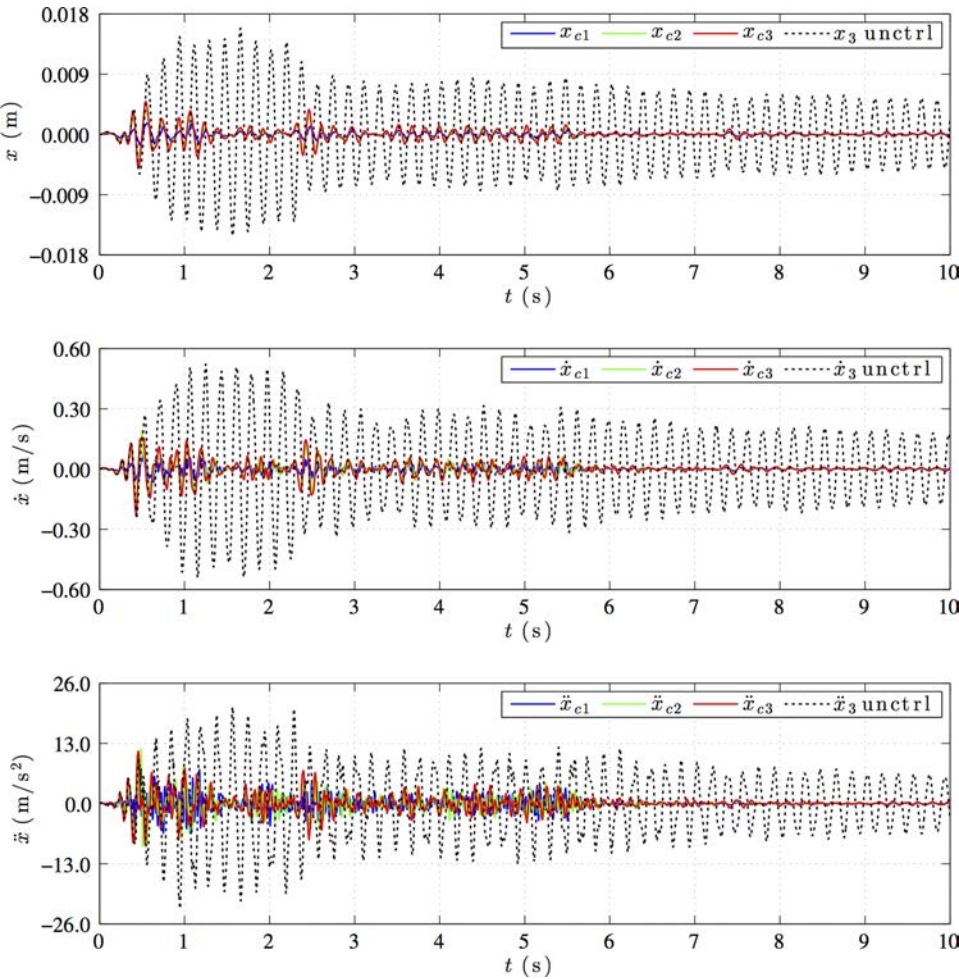


Fig. 13. Structural response with the fuzzy logic controller.

force generated during the control process does not need to be monitored, as happens in other controllers such as the clipped-optimal algorithm.

The damper force and the control signal of the semi-active control system are shown in Fig. 14. As can be seen, the proposed fuzzy controller provides intermediate levels of control current instead of the bi-state control signal used in many semi-active controllers allowing intermediate damping states over the full range of operation of the device. Generally, the results show that the proposed fuzzy logic controller is capable to determine with sufficient reliability the required control action to reduce the response of the 3-DOFs system.

The hysteretic behavior of the MR damper during the numerical simulation is also characterized in Fig. 15. As can be seen the proposed fuzzy based controller explores the operating range of the actuator and globally, the hysteretic loops are in line with

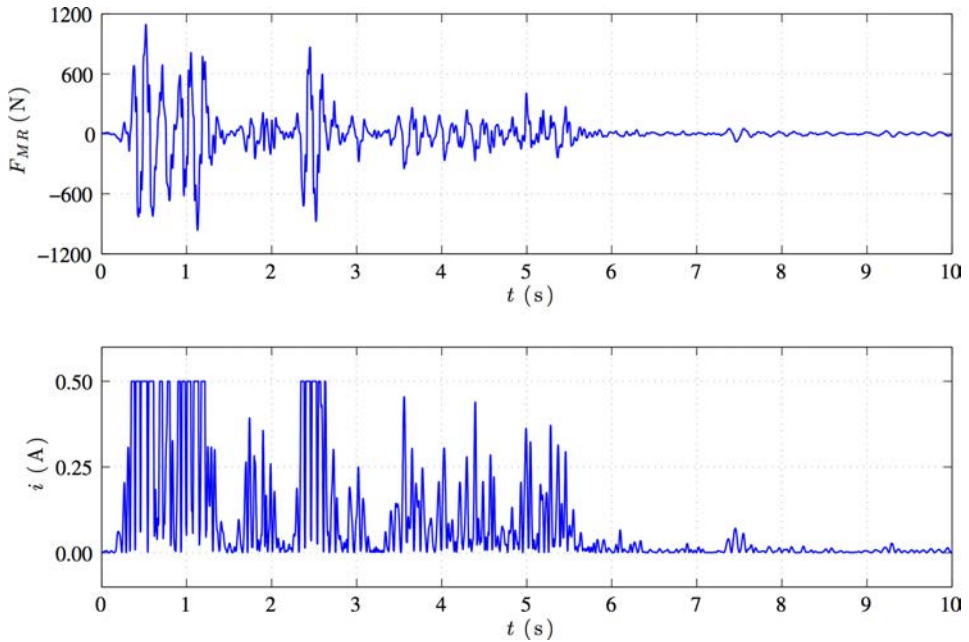


Fig. 14. Damper force and corresponding operating current (semi-active control).

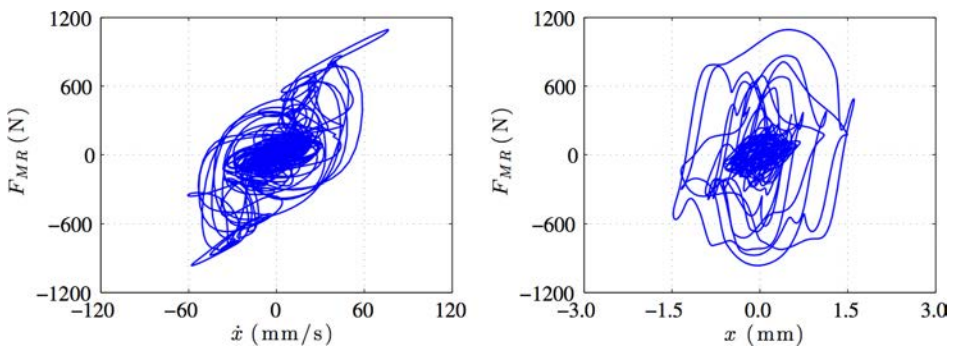


Fig. 15. MR damper force in a semi-active control mode (see specs in Sec. 2).

those found with the other controllers although in this case presenting a more irregular shape.

The main drawback of designing fuzzy controllers is related with the definition of the membership function parameters and the inference rules that relate the inputs with the desired control output. Structural systems usually include several sources of nonlinearities and/or uncertainties that hinder the development of simple control rules based on human knowledge about the system behavior. In these cases, soft computing techniques such as ANFIS or genetic algorithms (GAs) are most

appropriate to find the best set of fuzzy rules or adjustment of a set of fuzzy parameters in accordance with a given training data for a desired control action.

5. Performance Evaluation

The evaluation criteria are based on a comparison of the peak responses of each controlled system to those of the uncontrolled system and passive modes. The results

Table 2. Peak responses — *El-Centro* NS earthquake (0.2*t*).

Control strategy	$x(\text{cm})$	$\dot{x}(\text{cm/s})$	$\ddot{x}(\text{cm/s}^2)$	Drift (cm)	$f(\text{N})$
Uncontrolled	0.695	27.09	1305	0.695	—
	1.251	45.78	1736	0.581	
	1.587	54.02	2272	0.371	
Passive OFF	0.518 (0.75)	20.02 (0.74)	999 (0.77)	0.518 (0.75)	166.4
	0.907 (0.73)	34.51 (0.75)	1358 (0.78)	0.443 (0.76)	
	1.191 (0.75)	42.79 (0.79)	1791 (0.97)	0.292 (0.79)	
Passive ON	0.171 (0.25)	7.77 (0.29)	613 (0.47)	0.171 (0.24)	1048.9
	0.423 (0.34)	19.36 (0.42)	1066 (0.61)	0.253 (0.44)	
	0.560 (0.35)	25.58 (0.47)	1366 (0.60)	0.208 (0.56)	
ANFIS-FLC	0.164 (0.24)	7.07 (0.26)	739 (0.57)	0.164 (0.24)	909.8
	0.410 (0.33)	17.59 (0.38)	963 (0.55)	0.247 (0.43)	
	0.529 (0.33)	23.64 (0.44)	1285 (0.57)	0.194 (0.52)	

Table 3. Evaluation criteria for the controlled system.

Evaluation criteria	Description
$J_1 = \max_{t,i} \left(\frac{x_{ic}(t)}{x_{max,u}(t)} \right)$	Maximum peak floor displacement ratio whereby the floor displacements over time are normalized by the maximum peak uncontrolled displacement.
$J_2 = \max_{t,i} \left(\frac{\dot{x}_{ic}(t)}{\dot{x}_{max,u}(t)} \right)$	Maximum peak floor velocity ratio whereby the floor velocities over time are normalized by the maximum peak uncontrolled velocity.
$J_3 = \max_{t,i} \left(\frac{\ddot{x}_{ic}(t)}{\ddot{x}_{max,u}(t)} \right)$	Maximum peak floor acceleration ratio whereby the floor accelerations over time are normalized by the maximum peak uncontrolled acceleration.
$J_4 = \max_{t,i} \left(\frac{\ x_{ic}(t)\ }{\ x_{max,u}(t)\ } \right)$	Maximum RMS floor displacement ratio given in terms of the maximum RMS absolute displacement over time with respect to the uncontrolled case.
$J_5 = \max_{t,i} \left(\frac{\ \dot{x}_{ic}(t)\ }{\ \dot{x}_{max,u}(t)\ } \right)$	Maximum RMS floor velocity ratio given in terms of the maximum RMS absolute velocity over time with respect to the uncontrolled case.
$J_6 = \max_{t,i} \left(\frac{\ \ddot{x}_{ic}(t)\ }{\ \ddot{x}_{max,u}(t)\ } \right)$	Maximum RMS floor acceleration ratio given in terms of the maximum RMS absolute acceleration over time with respect to the uncontrolled case.
$J_7 = \max_{t,j} \left(\frac{\sum_j P_j(t)}{x_{max,u} W} \right)$	Maximum control power normalized by $(\dot{x}_{max} W)$, i.e., the weight of the structure times the maximum uncontrolled velocity. W is the total weight of the structure.
$J_8 = \max_{t,j} \left(\frac{\sum_j \int_0^t \frac{1}{t_f} P_j(t)}{x_{max,u} W} \right)$	Total power required to control the response of the structure normalized by the weight of the structure times the maximum uncontrolled velocity.

Table 4. Evaluation criteria for each control configuration.

Control mode	J_1	J_2	J_3	J_4	J_5	J_6	J_7	J_8
Passive OFF	0.750 112%	0.371 158%	0.789 31%	0.487 192%	0.495 140%	0.507 77%	0.021 -59%	0.0014 0%
Passive ON	0.353	0.144	0.602	0.167	0.206	0.287	0.051	0.0014
ANFIS – FLC	0.334 -5%	0.131 -9%	0.566 -6%	0.164 -2%	0.192 -7%	0.261 -9%	0.035 -31%	0.0014 -0%

of this analysis are summarized in Table 2. The results show the effectiveness of the proposed fuzzy based controller in reducing the response of the structure. In this case the fuzzy controller outperforms the passive control modes in almost all peak responses (with exception of the first floor acceleration, although with a significant reduction compared with the uncontrolled case). The results also show that using the MR damper in a semi-active control mode results in lower peak drifts compared with the passive ON configuration namely in the two upper floors.

A new set of evaluation criteria was used to evaluate the effectiveness of each control mode.^{10,11} These evaluation criteria are given in Table 3, including normalized and RMS responses and also control requirements.¹² The first three criteria (J_1 , J_2 , J_3) are based on the peak responses and the next four (J_4 , J_5 , J_6) are related with RMS (normed) structural responses. In these equations, $|\cdot|$ denotes the absolute value and $\|\cdot\|$ is the L_2 norm given by

$$\|\cdot\| = \sqrt{\frac{1}{t_f} \int_0^{t_f} [\cdot]^2 dt}, \quad (5.1)$$

where $t_f = t_{\max}$ represent the total excitation duration. The last three performance indices (J_7 , J_8) are intended to evaluate the effectiveness of the MR actuator. The results achieved with each control mode are presented in Table 4.

From the results presented in Tables 3 and 4, it can be concluded that the semi-active fuzzy controller in a comprehensive way outperforms the passive control configurations (note that lower values indicate a better performance). Note that the performance indices J_1 to J_6 are related with the structural responses, and reflect the higher efficiency of the proposed semi-active control system over the passive control modes. In fact, the performance indices are consistent with the results reported in the peak responses analysis.

6. Conclusion

This paper addressed the optimization of a fuzzy based controller using an ANFIS. This modeling approach was used to develop a neuro-fuzzy model for a MR damper based on artificial training data. A numerical simulation was used to obtain and analyze the effectiveness of the proposed semi-active controller in reducing the response of a simple structural system. It was verified that the semi-active controller

allowed a more efficient management of the actuator displaying a better performance in reducing the structural response than the passive control modes. Hence, it can be concluded that the proposed semi-active fuzzy based control system outperforms a fully passive system allowing a considerable reduction of lateral motion of the masses using only a MR damper in a non-collocated control configuration. Further research must be carried out to improve the fuzzy optimization procedure using ANFIS and also to validate the application of neuro-fuzzy controllers to a wide range of structural configurations.

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